

Hybrid Fuzzy-PID Control of a Nuclear Cyber-Physical System

Working Under Varying Environmental Conditions

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Abstract: In real-time control of Cyber-Physical Systems (CPSs), physical process variables are monitored and processed by intelligent controllers for keeping the values of safety parameters between given thresholds. Environmental conditions can affect the system dynamics and also the controller function. This paper presents a hybrid fuzzy-PID (Proportional-Integral-Derivative) controller, which learns the optimal PID controller parameters and adapts them according to the environmental parameters and process variables values.

The proposed intelligent controller is applied to respond to the changes and transients in the environmental inlet air temperature of the secondary loop of a Lead-Bismuth Eutectic eXperimental Accelerator Driven System (LBE-XADS), for keeping the secondary average coolant temperature in a range of values between two safety thresholds.

Keywords: Cyber-Physical System; Hybrid Fuzzy-PID Controller; Environmental Conditions; Nuclear Power Plants (NPPs); Lead-Bismuth Eutectic eXperimental Accelerator Driven System (LBE-XADS).

ABBREVIATIONS

CPS	Cyber-Physical System
NPP	Nuclear Power Plant
PID	Proportional-Integral-Derivative
LBE-XADS	Lead-Bismuth Eutectic eXperimental Accelerator Driven System
FLC	Fuzzy Logic Control

NOTATIONS

t	Time
$e(t)$	Residual between the measured value and the set point
K_p	Proportional gain
K_i	Integral gain
K_d	Derivative gain
$P(t)$	LBE-XADS thermal power
$Q(t)$	Proton energy
$T_{LB}^{C,P}$	Primary outlet coolant temperature
$T_{LB}^{P,C}$	Primary inlet coolant temperature
$T_{LB}^{ac,C}$	Average in-core temperature
$T_o^{in,S}$	Secondary inlet oil temperature
$T_o^{out,S}$	Secondary outlet oil temperature
$T_o^{av,S}$	Secondary average oil temperature
$\Gamma_a(t)$	Air mass flow rate
$T_o^{th,u}$	Upper threshold of the oil temperature
$T_o^{th,l}$	Lower threshold of the oil temperature
Ψ	Monotonically increasing function
$T_a^{in,S}$	Inlet air temperature
$T_o^{ref,S}$	Set point of secondary average oil temperature
dK_p	Change of the proportional gain
dK_i	Change of the integral gain
dK_d	Change of the derivative gain
x	Input of an input-output data pair
y	Output of an input-output data pair
f	Mapping function
\bar{e}_{SS}	Average steady-state drift
A_{SP}	Settling area
T_j	Fuzzy sets of the j -th Antecedent, $j=1,2,\dots,7$
I_j	Fuzzy sets of the j -th Consequent
R_j	j -th fuzzy rule
Y	Fuzzy set obtained from a fuzzy conclusion
μ_Y	Membership function

I. INTRODUCTION

A Cyber-Physical System (CPS) combines a cyber system with a physical system, integrating computational resources into physical processes in order to add new capabilities to the original stand-alone physical system and realize real-time monitoring, dynamic control and decision support. The benefit from such capabilities makes CPSs increasingly operated in transportation, energy, medical and health-care, and other applications [1]-[3]. In CPSs, the cyber computation and the physical processes are dependable, and interact with each other through multiple and distinct modes (e.g., embedded cyber controllers monitor and control the system physical variables, whilst physical processes affect, at the same time, the monitoring system and the computation units by wired or wireless networks [2], [4], [5]). This information can be intelligently manipulated to guarantee the system to be self-adaptive, robust and reliable with respect to components failures and changes of the environmental operational conditions [4], [6-8], [16]. For example, in [44] the flight control of parafoil systems is shown to be improved when wind conditions are duly taken into account, whereas in [45,46] adaptive speed control strategies of autonomous vehicles under various traffic conditions are proposed.

Adaptivity of CPS control rules can be achieved based on real-time monitoring of physical and environmental variables. Stability is the main concern for real-time monitoring of CPSs. To satisfactorily control the dynamical systems, it is basically required the changes in the observed behavior of the physical system in case of any small perturbations imposed by the controller to be kept at minimum [37].

Traditionally, Proportional-Integral-Derivative (PID) control systems have been used for retroacting to actuators the actions to be undertaken for real-time controlling the system (see Fig. 1) [8], [16], [40]. Digital PIDs have overcome analog PIDs [9] but still cannot intelligently adapt their parameters setting to changing environmental conditions: a predesigned control function is implemented by a command signal, to control the physical process variables, which, once measured, are compared with a set point that originates the residual $e(t)$ (see Fig. 1). A strong limitation of this traditional

control approach is that incorrect calibration of the PID parameters with respect to the current physical and environmental conditions may significantly degrade the control performance and robustness [9], [10]. Moreover, under this control scheme, the physical system may be led to dangerous operational conditions not only due to unforeseen changes of the environmental conditions but also due to human errors, malicious attacks through the network and poor/wrong database use in support to parameters setting (see Fig. 2).

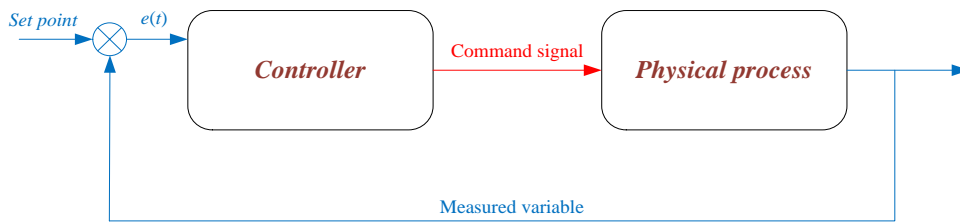


Fig. 1. The PID control scheme

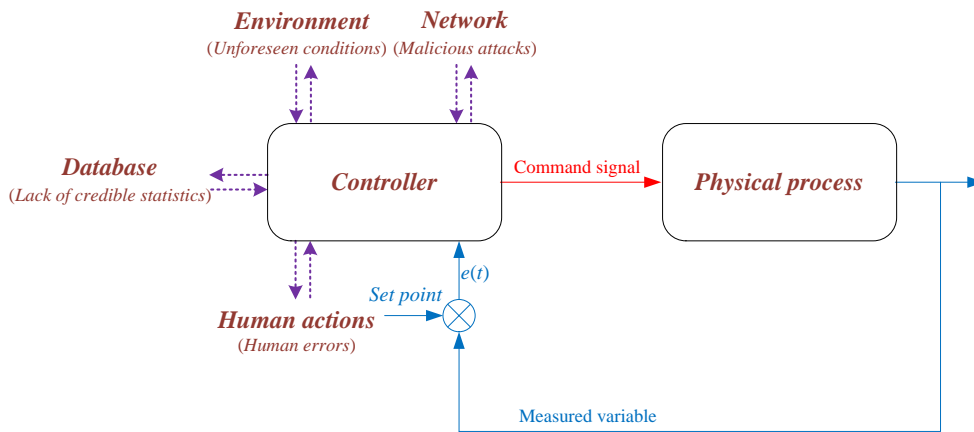


Fig.2. The cyber control scheme

To overcome the challenges of conventional PID control and add learning capabilities to a CPS controller, a Fuzzy Logic Control (FLC) strategy can be of help [14]. FLC has been increasingly used in real-world control engineering [15], [17], [39], [41], [47]. The fuzzy rules are obtained by a fuzzification of the ranges of the measured physical variables (Antecedents) and the logic mapping into the ranges of the control parameters (Consequents) [18], [19]. Nonlinear analysis has been adopted to build the FLC rules and improve the control performance, especially for systems operating under uncertain physical and environmental conditions [20]-[23]. In most cases, the fuzzy

rules are based on expert experience and knowledge of the system processes [24], [25]. The integration of a PID with a FLC (e.g., fuzzy-PID) is proven to be an intelligent, nonlinear and robust strategy that improves the performance of conventional PIDs, by readjusting the PID gains in an on-line and real-time manner [23], [42]. The resulting fuzzy-PID controller is, thus, aimed at accounting for time-dependent interactions of the cyber world with the physical world in an intelligent feedback loop, while at the same time, guaranteeing the stability of the CPSs, without causing the loss of control of the physical behaviors.

In practice, environmental conditions can play an important role on the physical process by triggering unforeseen system dynamics. For example, seawater is often used as the coolant of the secondary coolant circuit of Nuclear Power Plants (NPPs) and any seawater temperature change may affect the NPP cooling capability and, eventually, its performance and safety [11]; in Heating, Ventilation and Air-Conditioning (HVAC) systems, changes of the environmental temperature make the heat transfer coefficient and the energy balances change [8], [12], [13]. Thus, environmental parameters should be measured and integrated into the cyber real-time monitoring and control in an intelligent manner: the resulting CPS is expected to learn the optimal control settings in different environmental conditions, and self-adapt to the varying conditions to preserve safe operation of the physical system and avoid loss of control.

In this work, environmental parameters values are used as inputs of a fuzzy-PID controller that optimally sets the PID gains, denoted as K_p , K_i and K_d , for the proportional, integral, derivative parts of the control, respectively. By so doing, the hybrid fuzzy-PID is capable of controlling the CPS under varying environmental conditions. To increase the transparency of the hybrid fuzzy-PID control, the fuzzy rules are here optimized by resorting to a grid-type fuzzy partitioning approach [24], [26]-[29], where the fuzzy sets and the fuzzy rules are learnt from examples of input variables and optimal control settings.

The proposed controller is applied to respond to changes and transients of the environmental inlet air temperature of the secondary loop of a Lead-Bismuth Eutectic

eXperimental Accelerator Driven System (LBE-XADS) [30], so as to maintain the secondary average coolant temperature at a desired reference value, in-between two safety thresholds. The optimized CPS is finally compared with a traditional PID control scheme with fixed parameters and with a linearly variable gain PID control scheme.

The remainder of the paper is organized as follows. Section II presents in details the LBE-XADS that is taken as CPS case study. In Section III, an overview of the proposed hybrid fuzzy-PID control strategy is presented for the temperature control of the secondary cooling system of the LBE-XADS, under varying environmental conditions. The results of the proposed approach are compared with the results of the reference PID control in Section IV. Section V draws the conclusions of the work.

II. CPS CASE STUDY: THE LBE-XADS

A. The Physical Process

The Lead-Bismuth Eutectic eXperimental Accelerator Driven System (LBE-XADS) is a sub-critical, fast reactor in which the fission process for providing thermal power $P(t)$ is sustained by an external neutron source through spallation reaction by a proton beam $Q(t)$ accelerated by a synchrotron on a Lead-Bismuth Eutectic (LBE) target.

The primary cooling system is of pool-type with LBE liquid metal coolant leaving the top of the core, at full power nominal conditions, at temperature $\tau_{LB}^{C,P}$ equal to 400°C pushed by natural circulation enhanced by argon gas injection into the heat exchangers of the secondary cooling circuit and then re-entering the core from the bottom through the down-comer at temperature $\tau_{LB}^{P,C}$ equal to 300°C. The average in-core temperature of the LBE $T_{LB}^{ac,C}$ is taken as the mean of $\tau_{LB}^{C,P}$ and $\tau_{LB}^{P,C}$.

In the secondary cooling system, a flow of an organic diathermic oil is cooled in a secondary heat exchanger, from the inlet temperature $T_o^{in,S}$ equal to 320°C to the outlet temperatures $T_o^{out,S}$ equal to 280°C, at full nominal power of 80MWth. Thus, the average nominal temperature $T_o^{av,S}$ is 300°C. Cooling of the diathermic oil is obtained

through an air flow $\Gamma_a(t)$ (nominally equal to 750kg/s [43]) provided by three air coolers connected in series (i.e., secondary heat exchanger), that can provide a maximum air flow rate of 1000kg/s [30, 43].

B. The Control Strategy

In [30], the LBE-XADS was originally equipped with a simultaneous feedforward-feedback control (whose scheme is shown in Fig. 3) to control the system physical dynamics by maintaining $T_o^{av,S}$ close to 300°C during any operational transient and any design basis accident, for any LBE-XADS prototypical plant operated in any location in the World, where the air temperature ranges in $[0, 30]$ °C with an average value equal to 20°C [30]. On the contrary, an oil temperature beyond the upper threshold $T_o^{th,u}$ equal to 340°C would lead to degradation of its physical and chemical properties (i.e., high-temperature failure mode), whereas a temperature below the lower threshold $T_o^{th,l}$ equal to 280°C could result in thermal shocks for the primary fluid and, eventually, for the structural components (i.e., low-temperature failure mode) [30], [31]. The average temperature of diathermic oil $T_o^{av,S}$ is the controlled variable, and the air mass flow rate $\Gamma_a(t)$ is the command signal provided by the controller. As shown in Fig. 3, the feedback controller was a conventional PID with low values of the proportional (K_p), the integral (K_i) and the derivative (K_d) gains, and contributing for 70% of the control strategy; the remaining 30% was provided by the feedforward action obtained by a monotonically increasing function Ψ relating the required air mass flow rate $\Gamma_a(t)$ to the reactor power $P(t)$, according to Eq. (1):

$$\Gamma_a(t) = K_p e(t) + K_i \int e(t) dt + K_d \frac{de(t)}{dt} + \Psi(P(t)) \quad (1)$$

The three feedback terms of Eq. (1) are:

- 1) The Proportional (P)-term, that accounts for a control action proportional to the residual $e(t)$ (i.e., the difference between the desired set point $T_o^{ref,S}$ and the value of $T_o^{av,S}$ measured at time t) and that may lead to the large values of the steady-state residual over time in case of inaccurate setting;

- 2) The Integral (I)-term, that accumulates the past values of the residual over time, to reduce the steady-state residuals;
- 3) The Derivative (D)-term, that sets the control action based on the present slope of the residual curve over time, to improve the response to a dynamic transient.

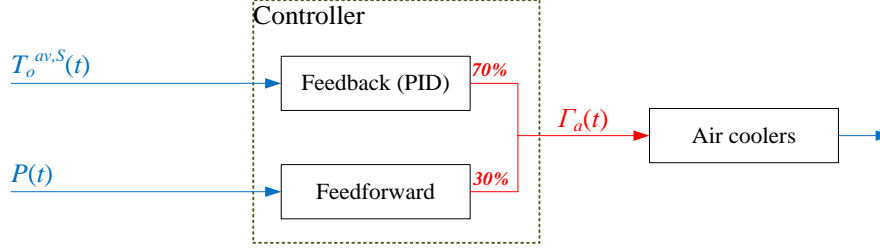


Fig. 3. Structure of the LBE-XADS control strategy

Fig. 4 shows the overall scheme of the LBE-XADS digital acquisition chain of environmental temperature $T_a^{in,S}$ and diathermic oil temperature $T_o^{av,S}$, that acts as the bounding layer between the physical process and the computation layer of the feedback controller. This digital acquisition chain is composed of two redundant channels (A and B) (see Fig. 4 shadowed box). Each channel consists of one signal sensor (S-A and S-B, measuring the average oil temperature $T_o^{av,S}$), one Bistable Processor Logic (BPL) subsystem (BPL-A and BPL-B), and one Local Coincidence Logic (LCL) subsystem (LCL-A and LCL-B). An independent signal sensor and an independent BPL is installed in each channel. Two redundant measured signals of $T_o^{av,S}$ are collected by S-A and S-B and processed by the corresponding BPL. The signal processing proceeds only if both channels produce the residual $e(t)$ compared with the set point $T_o^{ref,S}$: each $e(t)$ from a BPL is sent to both LCL-A and LCL-B, which process information by an “AND” gate. In other words, the residual signal is processed only when receiving two residual signals $e(t)$ originating from the two different BPLs; the information is, then, processed by an “OR” gate at the end of the digital acquisition chain. Once the residual signal $e(t)$ is processed, it is sent to the PID feedback.

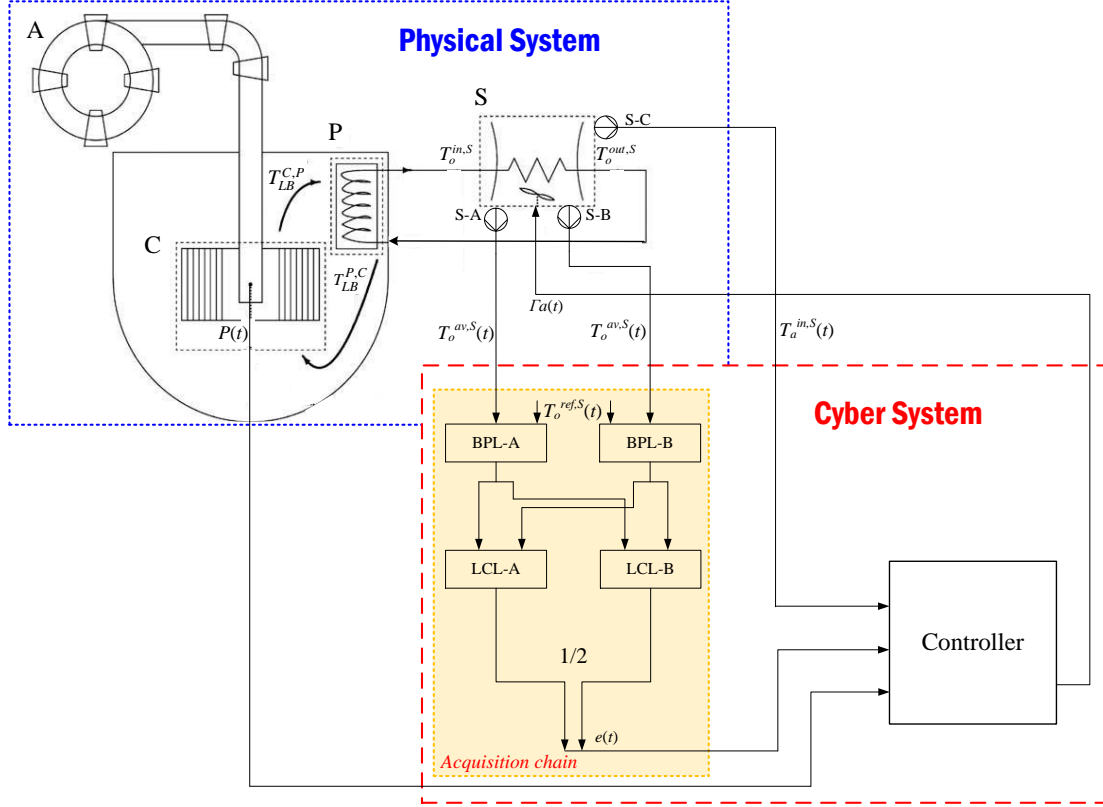


Fig. 4. The simplified scheme of the LBE-XADS with its digital acquisition chain

III. CONTROL STRATEGY UNDER VARYING ENVIRONMENTAL CONDITIONS

A. System Properties Identification

To show the effect of the environmental variable $T_a^{in,S}$ on the system dynamics and the corresponding control performance of the original PID, a Matlab/Simulink model of the LBE-XADS has been run $N_I=50$ times with a mission time t_M of 3000s. The inlet air temperature $T_a^{in,S}$, initially equal to 20°C, changes at time $t=1000$ s to a random value $T_{a,c}^{in,S}$ sampled from a uniform distribution $U\sim[0, 30]^\circ\text{C}$. The results are shown in Fig. 5 (solid lines): at time $t=1000$ s, the physical process rapidly responds to the change of the environmental condition (i.e., the inlet air temperature) and the average oil temperature $T_o^{av,S}$ deviates from its reference level (300°C) towards a new stationary temperature after approximately 800s, confirming the adaptive capability of the original PID.

For completeness, some (uncontrollable) scenarios (dashed lines) are also shown

where $T_o^{av,S}$ continues rising, eventually reaching the threshold of the high-temperature failure mode before the end of the mission time, because triggered by temperature changes that fall out of the design limits of the cooling system (e.g., for $T_o^{av,S} > 300^\circ\text{C}$ the maximum $\Gamma_a(t)$ equal to 1000kg/s is not capable of removing the decay heat). In light of this design limit of the LBE-XADS cooling system, in what follow we will discard such uncontrollable scenarios and focus on improving the controllability of the LBE-XADS under the nominal temperature range $[0, 30]^\circ\text{C}$.

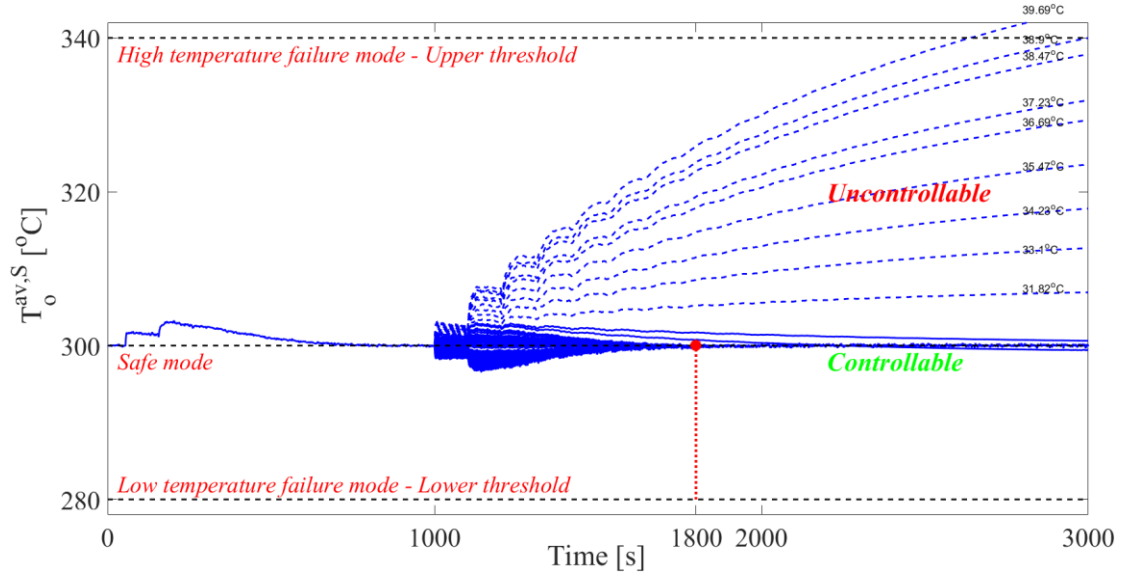


Fig. 5. The $N_1=50$ runs of the LBE-XADS controlled with the original PID when the inlet air temperature changes to a random value uniformly distributed in $[0, 30]^\circ\text{C}$ at time $t=1000\text{s}$

B. The Hybrid Fuzzy-PID Controller

A hybrid fuzzy-PID controller that automatically adapts the PID gains to different inlet air temperature $T_a^{in,S}(t)$ is here proposed. The structure of the controller is shown in Fig. 6. The idea is that the Fuzzy logic element “filters” the $T_a^{in,S}$ (Antecedent) and, based on the fuzzy logic reasoning, it provides the PID with the adaptation gain changes, dK_p , dK_i and dK_d (Consequents), that need to be added to the current values of the gains K_p , K_i and K_d of Eq. (1), in order to adapt to the changed environmental conditions.

The rules of the FLC are learnt by resorting to a grid-type fuzzy partitioning approach, as illustrated in the next subsection [24], [26]-[29], [32].

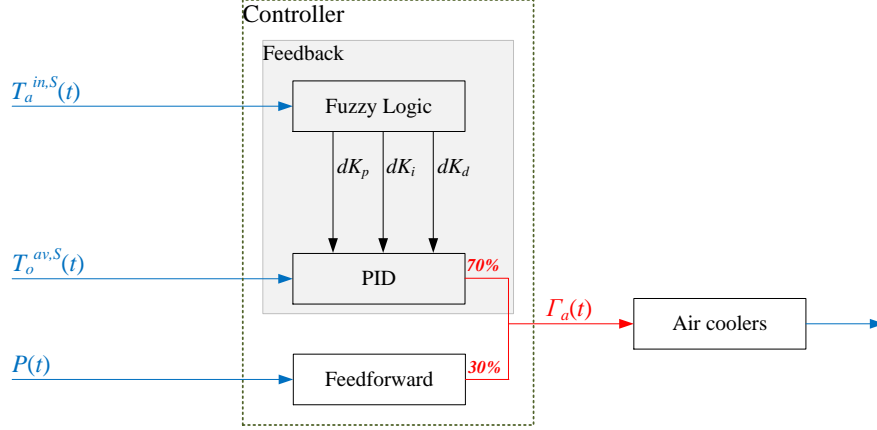


Fig. 6. Structure of the controller under variable inlet air temperature conditions

a. The Learning Algorithm

Fuzzy logic is used for combining a number of antecedents and consequents linguistic statements into relations by fuzzy connections, where a general grid-type fuzzy partitioning approach can be used to learn the fuzzy rules from the numerical data of the sampled input-output data pairs [26], [29], [32]. The objective of the grid-type fuzzy partitioning approach is to separate the input-output feature space into a set of uniform or non-uniform grids with predefined membership functions, and, then, to obtain the most transparent fuzzy rules linking the antecedents and consequents of the available examples. The key point of the approach is that the optimal input-output data pairs are those that can be univocally mapped in the feature space, which means that the visual connection between antecedents and consequents should be explicit. Taking a one-input-one-output FLC as illustrative example, suppose n optimal input-output data pairs are obtained from m pairs of examples:

$$\left(x^{(1)}, y^{(1)}\right), \left(x^{(2)}, y^{(2)}\right), \dots, \left(x^{(n)}, y^{(n)}\right) \quad (2)$$

where x and y are the input and the output, respectively. The n (x,y) data pairs can be plotted in their feature space. If the data pairs show a clear mapping function $f: x \rightarrow y$ (see Fig. 7(a)), decision makers can determine a set of fuzzy rules with predefined membership functions, according to their knowledge, whereas when the data pairs scatter in the feature space (see Fig. 7(b)), and the connection between x and y is implicit, the fuzzy sets and fuzzy rules cannot be univocally generated.

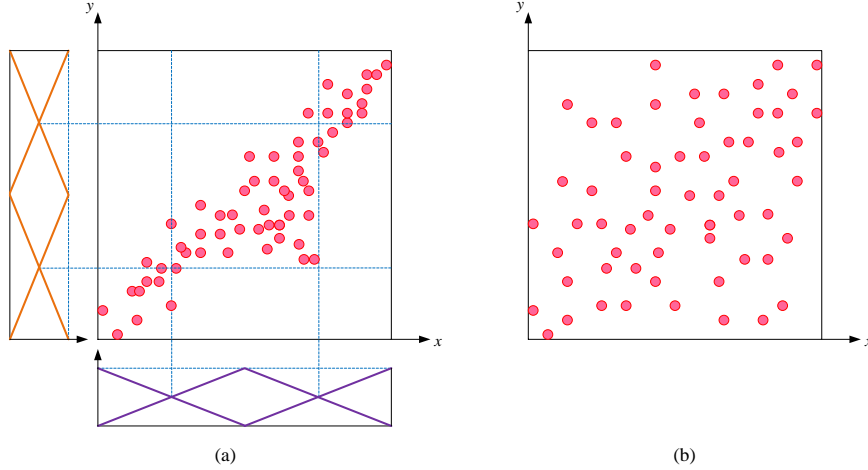


Fig. 7. Illustration of the general grid-type fuzzy partitioning approach: (a) optimal input-output data pairs, presenting a clear relationship in the feature space, to generate univocal fuzzy sets and fuzzy rules; (b) data pairs scattered in the feature space.

b. Fuzzy Rules Generation for the Hybrid Fuzzy-PID Controller

The generation of the fuzzy sets and rules from examples of input variables and optimal control settings follows two steps as for the grid-type fuzzy partitioning approach. Step 1 collects the optimal input-output data pairs from the randomly sampled examples and extracts the optimal pairs among the collected examples on the basis of two performance indexes. Step 2 generates the fuzzy sets and the fuzzy rules for the learnt significant input-output data pairs.

Step 1: Collection of the input-output data pairs.

We define two performance indexes to be used for extracting the optimal pairs with respect to one trial of the simulation of the LBE-XADS (controlled with the original PID) in which the inlet air temperature $T_a^{in,S}$ changes to a random value uniformly distributed in $[0, 30]^\circ\text{C}$, at $t=1000\text{s}$. The profile of the average oil temperature $T_o^{av,S}$ within the mission time of 3000s is shown in Fig. 8. When $T_a^{in,S}$ changes to a random value $T_{a,C}^{in,S}$ at $t=1000\text{s}$, the physical process rapidly responds to the change of environmental condition. $T_o^{av,S}$ recovers from the step response after a resettling time roughly equal to 800s , and the system dynamics reaches a new steady state.

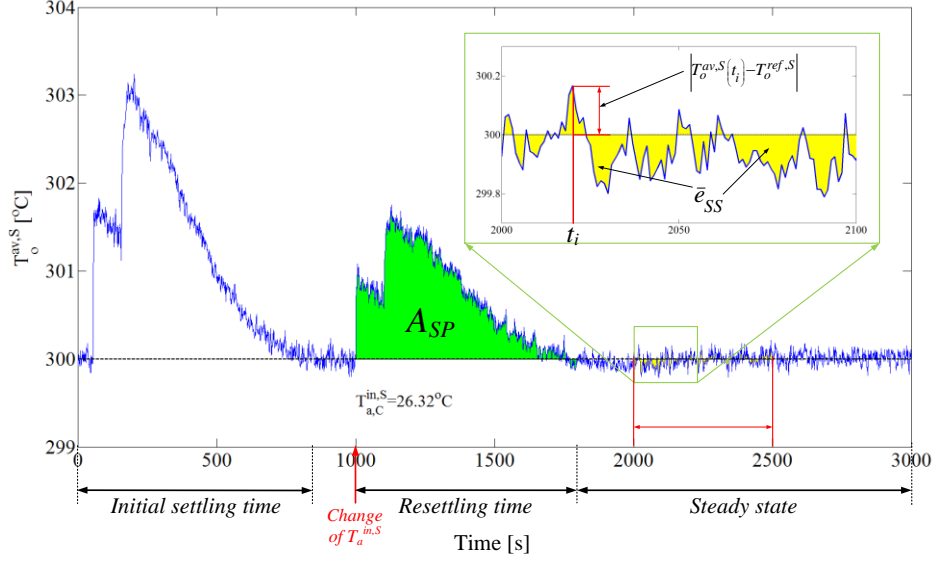


Fig. 8. Illustration of the characterization indexes

As it is known, in the PID controller, the steady-state residual error is inversely proportional to K_p , whose inappropriate value renders the control unstable, whereas K_d limits the high-frequency noise at the system steady state. Thus, the performances of K_p and K_d are to be evaluated by an average steady-state drift (\bar{e}_{SS}) defined in Eq. (3) below, that is the average residual of $T_o^{av,S}$ with respect to $T_o^{ref,S}$ after the system reaches a new steady-state condition. To obtain the optimal $(T_{a,C}^{in,S}, dK_p)$ and $(T_{a,C}^{in,S}, dK_d)$ data pairs to be used for the learning algorithm of Section III.B.a, \bar{e}_{SS} has to be minimized:

$$\bar{e}_{SS} = \frac{\sum_{i=1}^P |T_o^{av,S}(t_i) - T_o^{ref,S}|}{P} \quad (3)$$

where, the time interval $[t_l, t_p]$ is the period of time after $T_o^{av,S}$ reaches steady state and, P is the number of the temperature measurements within the period. In this study, $[t_l, t_p]$ is taken equal to $[2000, 2500]$ s and, thus, $P=501$. Notice that $\bar{e}_{SS}=0.10^\circ\text{C}$ is taken as the largest acceptable value of the average steady-state bias, taking into account the value of 0.07°C of the original model at full power, nominal condition and the uncertainty in the sampling. On the other side, it is also known that K_i drives the PID adjustment towards the steady state by accumulating the past values of the residual over time to reduce the steady-state residuals. Thus, the better tuned is the K_i value the better

the PID responds to the environmental change of $T_a^{in,S}$ in shorter settling time. A settling area A_{SP} is defined in Eq. (4) to quantify the accumulated residuals over (t_{AC} , t_{SP}) due to the change of inlet air temperature, and has to be minimized to obtain the optimal ($T_{a,C}^{in,S}$, dK_i) data pairs from examples:

$$A_{SP} = \int_{t_{AC}}^{t_{SP}} |T_o^{av,S}(t) - T_o^{ref,S}| dt \quad (4)$$

where, t_{AC} is the starting change time of $T_a^{in,S}$, i.e., $t_{AC}=1000$ s in our case, and t_{SP} is the end time of the PID settling period, i.e., $t_{SP}=1800$ s (as suggested by the scenarios plotted in Fig. 5). At full power nominal conditions, the average drift within the time interval of [1000, 1800]s is 0.067°C and the accumulated residual is, thus, equal to 53.6. For the conservative consideration of the optimal data pairs, 300 is assumed as the acceptable value of A_{SP} .

In this step, three groups of $n_2=1000$ simulations each of the LBE-XADS are run, each one devised to obtain the input-output data pairs, for finding the optimal dK_p , dK_i , and dK_d , respectively:

1) *Group 1:*

- a. Set the initial inlet air temperature $T_a^{in,S}(0)$ equal to 20°C and start the simulation.
- b. At $t=1000$ s, sample a random value of $T_{a,C}^{in,S}$ uniformly distributed in [0, 30]°C and the value of the change of the gain K_p from a uniform distribution $U\sim[-50, 50)$, whereas, the K_i and K_d are kept equal to the reference values 0.2 and 0, respectively; continue the present simulation till the mission time $t_M = 3000$ s.
- c. Calculate the value of the performance index, \bar{e}_{SS} , and represent its values on the three-dimensional diagrams, i.e., ($T_{a,C}^{in,S}$, dK_p , \bar{e}_{SS}). When the calculated value is smaller than 0.07°C, the input-output data pair is labelled as optimal to be utilized hereafter.

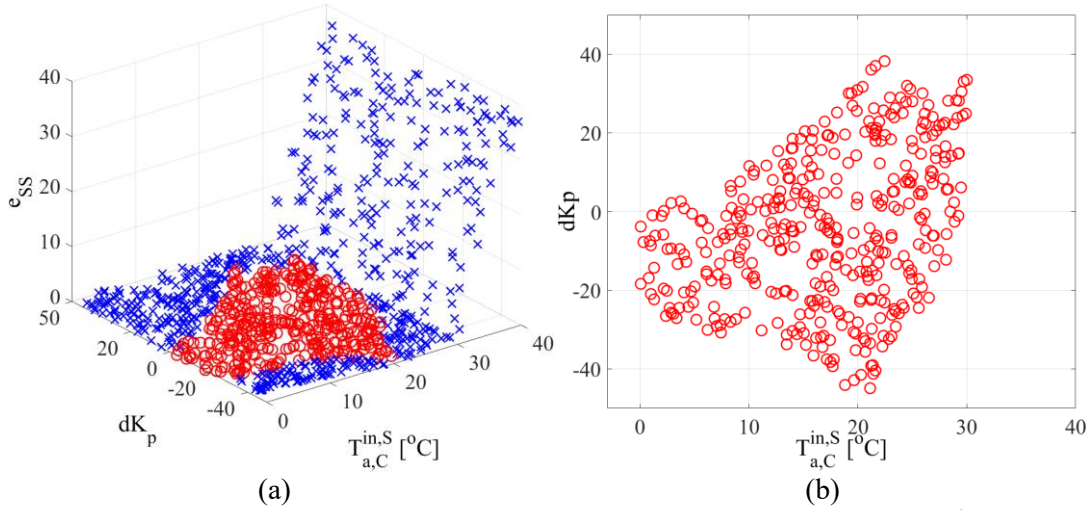


Fig. 9. Results of the $n_2=1000$ runs for the Group 1: (a) three-dimensional plot of $(T_{a,C}^{in,S}, dK_p, \bar{e}_{SS})$, where circles are the value of \bar{e}_{SS} smaller than 0.07°C and crosses are the value of \bar{e}_{SS} exceeding 0.07°C ; (b) optimal $(T_{a,C}^{in,S}, dK_p)$ data pairs (circles of Fig. 9(a))

Fig. 9(a) shows that the value of \bar{e}_{SS} is smaller than 0.07°C if the inlet air temperature changes to values smaller than 30°C . The sparsity of the optimal input-output data pairs $(T_{a,C}^{in,S}, dK_p)$ (circles in Figs. 9(a) and (b)) suggests the lack of a univocal mapping correlation of K_p with changes of $T_a^{in,S}$ and, thus, defines the weakness of K_p in controlling the physical dynamics under variable environmental conditions.

Whereas, if $T_{a,C}^{in,S}$ is larger than 30°C (i.e., out of the system design operational limits), \bar{e}_{SS} exceeds by far the acceptable value of 0.07°C , such that no optimal input-output data pairs can be expected. As already said, this phenomenon is due to the fact that the maximum air mass flow rate $\Gamma_{a,max}$ cannot meet the required value when $T_{a,C}^{in,S}$ exceeds 30°C .

2) Group 2:

- a. Set the initial inlet air temperature $T_a^{in,S}(0)$ equal to 20°C and start the simulation.
- b. At $t=1000\text{s}$, sample a random value of $T_{a,C}^{in,S}$ uniformly distributed in $[0, 30]^\circ\text{C}$ and the value of the change of the gain K_i from a uniform distribution $U(-0.2, 0.4)$, whereas the K_p and K_d are kept equal to the reference values 50 and 0,

respectively; continue the present simulation till the mission time $t_M = 3000s$.

- c. Calculate the value of the performance index, A_{SP} , and represent the values of the performance index on the three-dimensional diagrams, i.e., $(T_{a,C}^{in,S}, dK_i, A_{SP})$.

When the calculated value is smaller than 300, the input-output data pair is labelled as optimal and stored for future utilization.

Fig. 10(a) shows the values of A_{SP} increasing with (i) increasing dK_i and decreasing $T_{a,C}^{in,S}$, or (ii) decreasing dK_i and increasing $T_{a,C}^{in,S}$. When the sampled $T_{a,C}^{in,S}$ is larger than 30°C (i.e., out of the system design operational limits), the value of the calculated A_{SP} is much larger than 300, no matter the value of dK_i .

Fig. 10(b) shows the optimal $(T_{a,C}^{in,S}, dK_i)$ (i.e., whose A_{SP} is below the acceptable value) and the existence of an explicit mapping relationship between dK_i and $T_{a,C}^{in,S}$. This makes this value crucial for adjusting the hybrid fuzzy-PID gains to the varying environmental conditions and controlling the system dynamics.

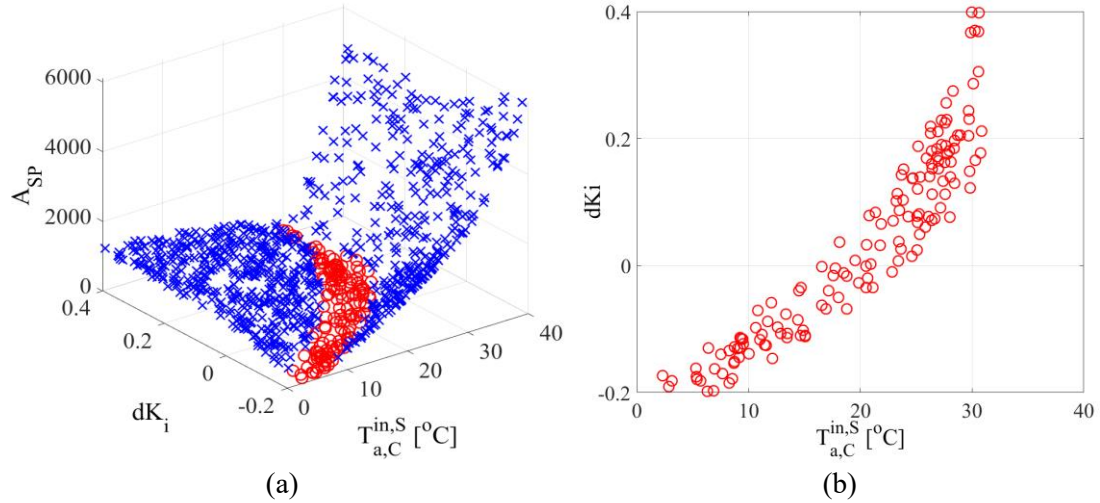


Fig. 10. Results of the $n_2=1000$ runs for the Group 2: (a) three-dimensional plot of $(T_{a,C}^{in,S}, dK_i, A_{SP})$, where circles are the value of A_{SP} smaller than 300 and crosses are the value of A_{SP} exceeding 300; (b) mapping of the optimal $(T_{a,C}^{in,S}, dK_i)$ data pairs

3) Group 3:

- a. Set the initial inlet air temperature $T_a^{in,S}(0)$ equal to 20°C and start the simulation.

- b. At $t=1000s$, sample a random value of $T_{a,C}^{in,S}$ uniformly distributed in $[0, 30]^\circ C$ and the value of the change of the gain K_d from a uniform distribution $U(-0.2, 0.4)$, whereas the K_p and K_i are kept equal to the reference values 50 and 0.2, respectively; continue the present simulation till the mission time $t_M = 3000s$.
- c. Calculate the value of the performance index, \bar{e}_{SS} , and represent values on the three-dimensional diagrams, i.e., $(T_{a,C}^{in,S}, dK_d, \bar{e}_{SS})$. When the calculated value is smaller than $0.07^\circ C$, the input-output data pair is labelled as optimal to be utilized hereafter.

Fig. 11(a) shows that the value of \bar{e}_{SS} is smaller than $0.07^\circ C$ if the inlet air temperature changes to values smaller than $30^\circ C$. The sparsity of the optimal input-output data pairs $(T_{a,C}^{in,S}, dK_d)$ (circles in Figs. 11(a) and (b)) suggests the lack of a univocal mapping correlation of K_d with changes of $T_a^{in,S}$ and, thus, defines the weakness of K_d in controlling the physical dynamics under variable environmental conditions.

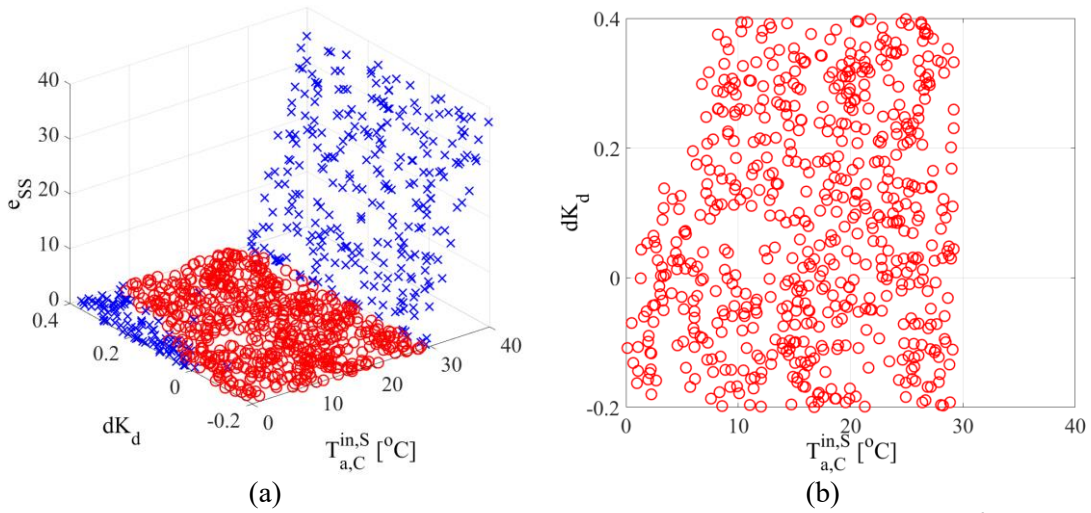


Fig. 11. Results of the $n_2=1000$ runs for the Group 3: (a) three-dimensional plot of $(T_{a,C}^{in,S}, dK_d, \bar{e}_{SS})$, where circles are the value of \bar{e}_{SS} smaller than $0.07^\circ C$ and crosses are the value of \bar{e}_{SS} exceeding $0.07^\circ C$; (b) mapping of the optimal $(T_{a,C}^{in,S}, dK_d)$ data pairs (circles in Fig. 11(a))

Whereas, with the fixed initial value of inlet air temperature $T_a^{in,S}(0)$ equal to $20^\circ C$, if $T_{a,C}^{in,S}$ is larger than $30^\circ C$ (i.e., out of the system design operational limits), \bar{e}_{SS}

exceeds by far the acceptable value of 0.07°C , such that no optimal input-output data pairs can be expected. Such a phenomenon is probably due to the fact that the maximum air mass flow rate $\Gamma_{a,max}$ cannot meet the required value when $T_{a,C}^{in,S}$ exceeds 30°C .

In conclusion, we can claim that, for this case study, K_p and K_d play a negligible role in the adaptation of the controller when working under varying environmental conditions, whereas K_i is sensitive to changes of the environmental conditions and, therefore, an appropriate setting of K_i to different $T_{a,C}^{in,S}$ can improve the adaptive properties of the controller for maintaining $T_o^{av,S}$ at the desired reference value. As a result, the controller block of Fig. 6 is transformed to that shown in Fig. 12.

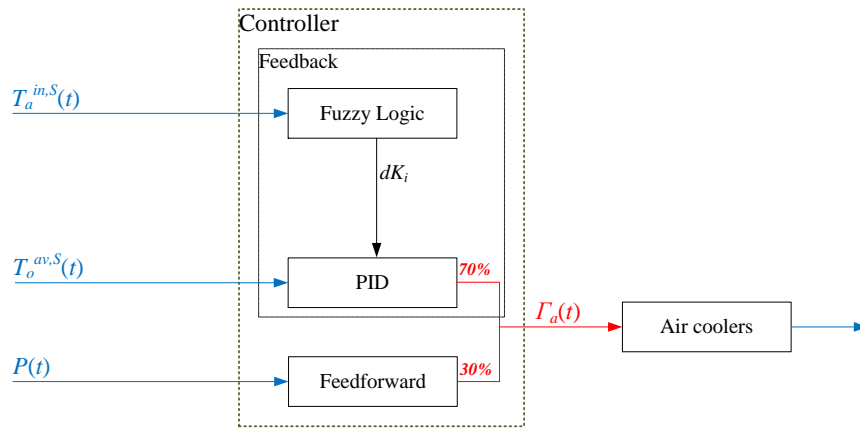


Fig. 12. Optimized structure of the controller under variable inlet air temperature

Step 2: Generation of the fuzzy sets and fuzzy rules

The grid-type fuzzy partitioning approach is used to generate the fuzzy sets and fuzzy rules for the fuzzy reasoning between the input $T_{a,C}^{in,S}$ and the output dK_i , as shown in Fig. 13:

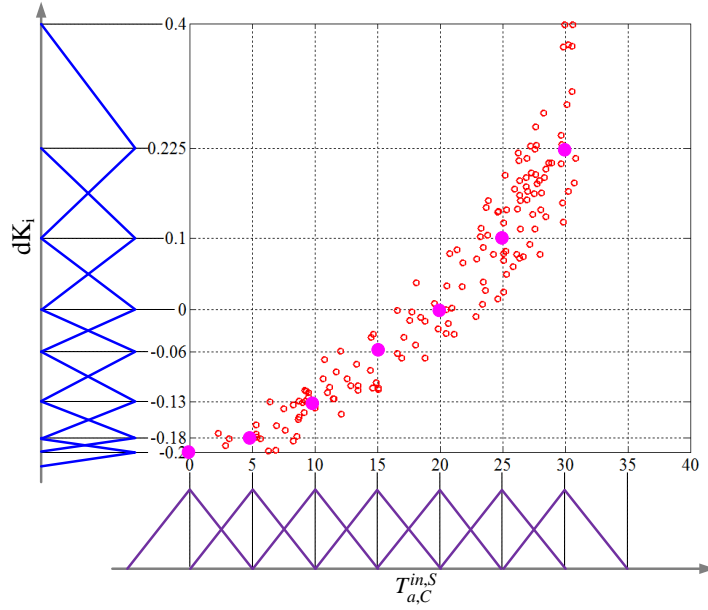


Fig. 13. Generation of the fuzzy sets by the grid-type fuzzy partitioning approach

- 1) Evenly partition the antecedents domain $T_{a,C}^{in,S}=[0, 30]^{\circ}\text{C}$ into reasonably small sets with given triangular membership functions; we partitioned the input support in seven sets $T_j, j=1,2,\dots,7, [-5, 0, 5], [0, 5, 10], [5, 10, 15], [10, 15, 20], [15, 20, 25], [20, 25, 30], [25, 30, 35]^{\circ}\text{C}$ (see Fig. 14) (See the Appendix for a sensitivity analysis that justifies the choice of the number of the fuzzy sets).
- 2) Collect $(T_{a,C}^{in,S}, dK_i)$ of the seven central points of the optimal sets.
- 3) Generate triangular output fuzzy sets $I_j, j=1,2,\dots,7$, whose largest degree of membership is centered on the dK_i of the data pairs $(T_{a,C}^{in,S}, dK_i)$ of Step (2). The seven output fuzzy sets, then, are $[-0.22, -0.2, -0.18], [-0.2, -0.18, -0.13], [-0.18, -0.13, -0.06], [-0.13, -0.06, 0], [-0.06, 0, 0.1], [0, 0.1, 0.225], [0.1, 0.225, 0.4]$ (see Fig. 14).
- 4) Define the fuzzy rules $R_j, j=1,2,\dots,7$, that define how to match antecedents and consequents, that, in our case, take the form:

$$R_j: \text{IF } (T_{a,C}^{in,S} \text{ belong to } T_j) \text{ THEN } (dK_i \text{ belong to } I_j).$$

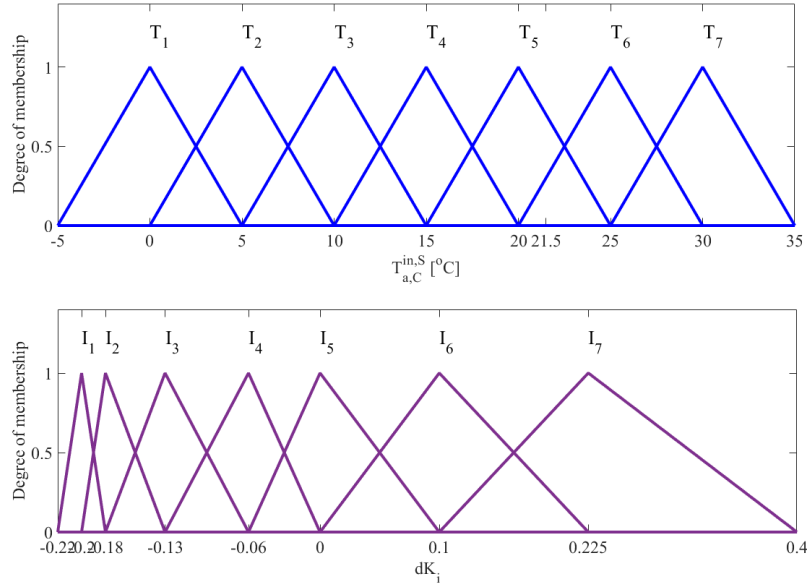


Fig. 14. The fuzzy sets generated from the grid-type partitioning approach

Defuzzification of the consequents is performed by the centroid calculation of a Mamdani-style fuzzy system inference [33]-[35]. The fuzzy conclusion of the consequent $y(dK_i)$ corresponding to an antecedent is a fuzzy set Y constituted by the membership function $\mu_Y(y(dK_i))$. Fig. 15 shows an example of fuzzy conclusion.

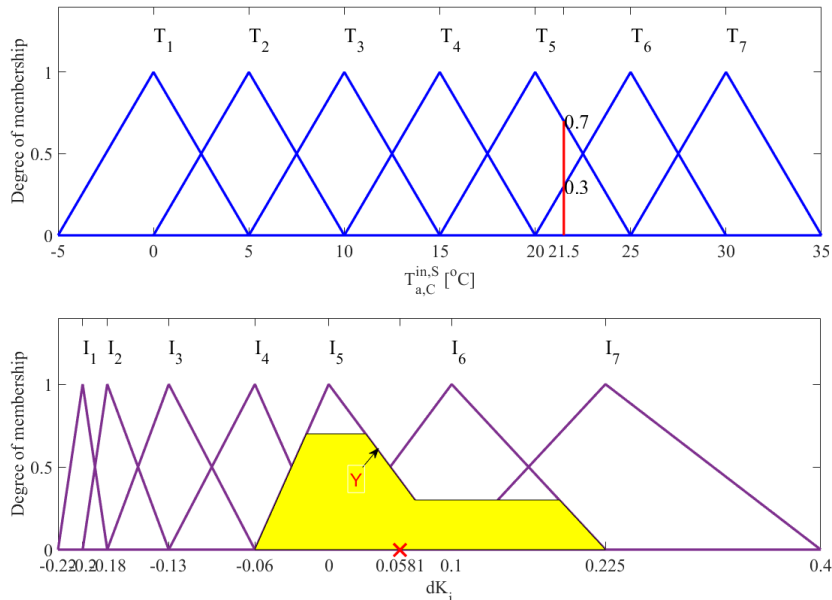


Fig. 15. Example of the consequent dK_i calculation: the value 0.0581 is the centroid of the weighted area of the two fuzzy sets activated by the antecedent $T_{a,c}^{in,S} = 21.5^\circ\text{C}$

For example, $T_a^{in,S}$ changes to 21.5°C at time $t=1000\text{s}$ and yields the fuzzy

membership values 0.7 in T_5 and 0.3 in T_6 , respectively:

$$T_{a,C}^{in,S(21.5)}: (0.7 \text{ in } T_5, 0.3 \text{ in } T_6).$$

Thus, the weights 0.7 and 0.3 are applied to T_5 and T_6 given for the antecedent $T_{a,C}^{in,S(21.5)}$.

The aggregated output fuzzy set Y is generated with the weights 0.7 in I_5 and 0.3 in I_6 (see shadowed area in Fig. 15), whose centroid is equal to 0.0581 that is assumed as consequent dK_i to a change of $T_{a,C}^{in,S}$ to 21.5°C.

C. The Variable-Gain PID Controller

In this Section, a variable-gain PID controller of literature [38] is defined for comparing its controlling capability with that of the proposed hybrid fuzzy-PID controller. In Section II.A, it has been pointed out that K_i is sensitive, whereas, K_p and K_d are insensitive to changes of the environmental conditions, therefore, the PID controller will only vary the K_i as in Eq. (5) [38].

Notice that the pivotal values of $T_{a,C}^{in,S}$ used to build the best fitting linear interpolation curve are (15°C, $K_i=0.135$) and (25°C, $K_i=0.30$), that are two optimal data pairs of Fig. 10(b) defining a likely interval of temperatures under which the plant is operated. Thus, the expression of K_i can be as (see Fig. 16):

$$K_i(T_{a,C}^{in,S}) = \begin{cases} 0.003, & 0^\circ\text{C} \leq T_{a,C}^{in,S} \leq 7^\circ\text{C} \\ 0.165 \cdot (T_{a,C}^{in,S} - 25) + 0.3, & 7^\circ\text{C} < T_{a,C}^{in,S} \leq 30^\circ\text{C} \end{cases} \quad (5)$$

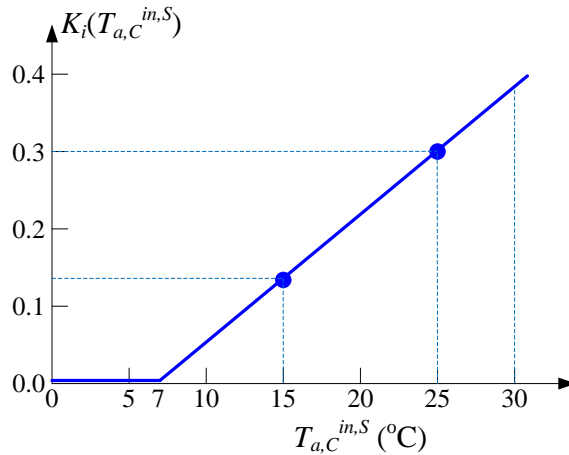


Fig. 16. Generation of the expression of K_i in terms of $T_{a,C}^{in,S}$

IV. RESULTS

In Section IV.A, the capability of the hybrid fuzzy-PID of Section III is benchmarked with the originally proposed PID controller with respect to a batch of sample simulations among the n_2 (described in Section III.A) that were used to train the algorithm for learning the rules. The practical usefulness of the proposed hybrid fuzzy-PID controller is, then, demonstrated with respect to a realistic scenario of daily cycle of inlet air temperatures, under normal environmental conditions and also transients due to wrong measurements of the temperature sensor (in Sections IV.B and IV.C, respectively).

A. Test on the Training Examples

We sample 18 random trials from the n_2 runs of the LBE-XADS Simulink model that were generated to train the rule generation algorithm, as described in Section III.B. As shown in Fig. 17(a), despite that the performance of the original PID controller (line with triangles) and the linearly variable K_i PID controller (line with diamonds) are acceptable for all of the 18 samples, the settling areas A_{SP} obtained with the hybrid fuzzy-PID controller (line with dots) are in most cases smaller than those obtained with the original PID and with the linearly variable K_i PID (also can be seen in Table I): in general terms, the optimized hybrid fuzzy-PID smooths down the step response caused by the change of the environmental conditions. At the same time, if $T_a^{in,S}$ changes to a random value within the interval of $[8, 30.5]^\circ\text{C}$, the average steady-state drift \bar{e}_{SS} is smaller with the hybrid fuzzy-PID (line with dots) than with the original PID (line with triangles) and with the linearly variable K_i PID (line with diamonds), as shown in Fig. 17(b). In particular, the larger the temperature, the better the performance of the hybrid fuzzy-PID with respect to the original PID (see cases number 17 and 18 with $T_{a,C}^{in,S}$ equal to 29.98°C and 30.45°C , respectively), not only because \bar{e}_{SS} decreases (from 0.157 to 0.0405 and from 1.16 to 0.0490, respectively), but also because the hybrid fuzzy-PID is capable of bringing back the secondary average coolant temperature to the reference value equal to 300°C , whereas the original PID cannot.

It is worth pointing out that neither the hybrid fuzzy-PID nor the original PID can safely control the system when $T_a^{in,S}$ changes to a value larger than 31°C , as mentioned in Section III.B.b.

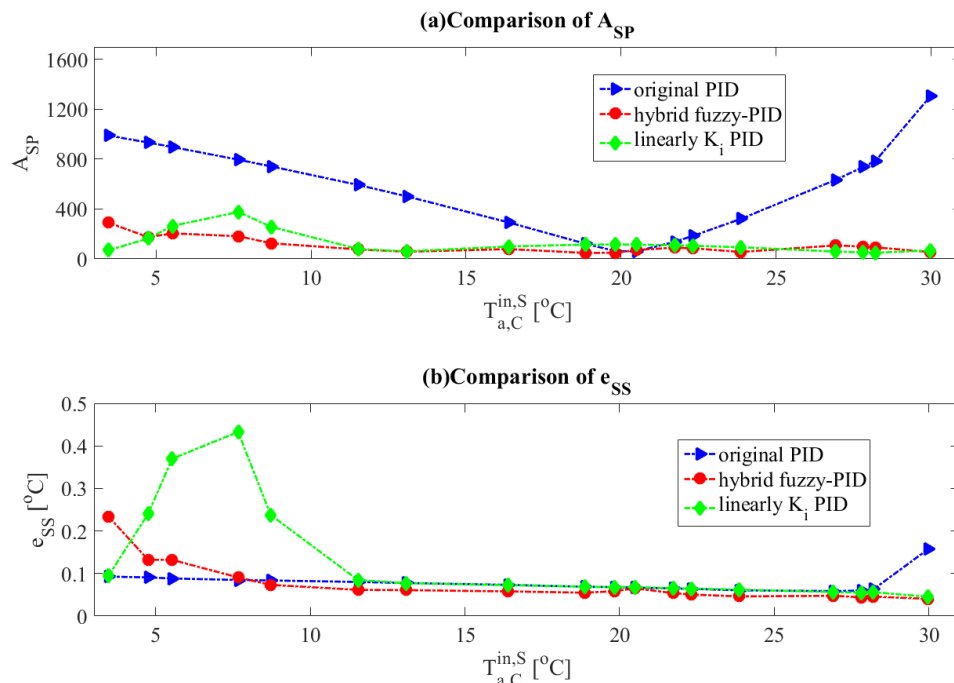


Fig. 17. Comparison between results of samples of the $T_o^{av,S}$ evolution with the optimized hybrid fuzzy-PID controller (line with dots), with linearly variable K_i PID (line with diamonds) and with the original PID (line with triangles): (a) comparison of A_{SP} ; (b) comparison of \bar{e}_{SS}

TABLE I

comparison of the performance indexes obtained from samples of the $T_o^{av,S}$ evolutions with the hybrid fuzzy PID, the linear variable K_i PID and the original PID

No.	$T_{a,c}^{in,S}$ (°C) at $t=1000s$	Original PID		Linear K_i PID		Hybrid fuzzy-PID	
		A_{SP}	\bar{e}_{SS} (°C)	A_{SP}	\bar{e}_{SS} (°C)	A_{SP}	\bar{e}_{SS} (°C)
1	3.474	989.7	0.0929	69.64	0.0947	290.8	0.233
2	4.751	934.1	0.0907	163.8	0.241	173.9	0.134
3	5.531	898.1	0.0878	264.4	0.370	205.1	0.133
4	7.663	796.0	0.0851	376.2	0.433	181.3	0.0913
5	8.716	742.6	0.0848	257.1	0.238	125.3	0.0742
6	11.54	592.2	0.0806	78.54	0.0848	76.14	0.0618
7	13.09	501.8	0.0777	60.74	0.0773	57.82	0.0607
8	16.38	293.1	0.0734	99.94	0.0727	79.02	0.0573
9	18.87	120.2	0.0688	114.7	0.0689	47.86	0.0550
10	19.83	62.53	0.0685	115.7	0.0680	49.84	0.0582
11	20.50	61.61	0.0680	114.2	0.0682	71.95	0.0651
12	21.73	136.8	0.0640	109.0	0.0654	90.43	0.0538
13	22.33	184.9	0.0833	105.3	0.0645	84.63	0.0510
14	23.87	321.3	0.0597	93.33	0.0621	55.57	0.0456
15	26.92	634.0	0.0583	59.02	0.0565	108.7	0.0474
16	28.21	786.1	0.0631	51.35	0.0550	93.02	0.0461
17	29.98	1305	0.157	49.07	0.0561	54.39	0.0405
18	30.45	1685	1.16	68.50	0.0454	43.21	0.0490

B. A Realistic Daily Air Temperature Scenario

For demonstration purpose, let us assume a realistic daily $T_a^{in,S}$ fluctuation as that in Fig. 18, where the initial value $T_a^{in,S}(0)$ is equal to 20°C corresponding to the nominal operating condition of the LBE-XADS and, the actual $T_a^{in,S}$ is fed to the Simulink model of the LBE-XADS, to test the hybrid fuzzy-PID controller working under real-time varying environmental conditions, at each 100s (i.e., smaller than the original PID resettling period, equal to 800s as shown in Fig. 5). This is done with the purpose of illustrating the capability of the hybrid fuzzy-PID to reduce residual $e(t)$ even when the system is fed with high frequency changes of $T_a^{in,S}$ that do not allow it to reach a new steady state before a new measurement is collected. Under these assumptions, Fig. 19 shows that $T_o^{av,S}$ can be kept close to the desired reference level of 300°C with both the hybrid fuzzy-PID controller (line with dots), the linear variable K_i PID controller (line with diamonds) and the original PID (line with triangles).

The proposed hybrid fuzzy-PID feedback controller reduces the drift $\bar{e}_{SS}=0.1376^\circ\text{C}$ of the original PID and $\bar{e}_{SS}=0.1027^\circ\text{C}$ of the variable K_i PID to $\bar{e}_{SS}=0.0600^\circ\text{C}$, showing that in this case the hybrid fuzzy-PID controller performs better than the original PID controller, under varying environmental conditions.

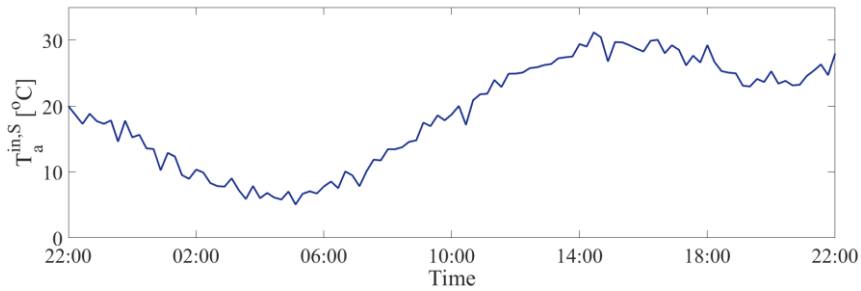


Fig. 18. Realistic daily $T_a^{in,S}$ fluctuation pattern, embedded into the simulation of the LBE-XADS

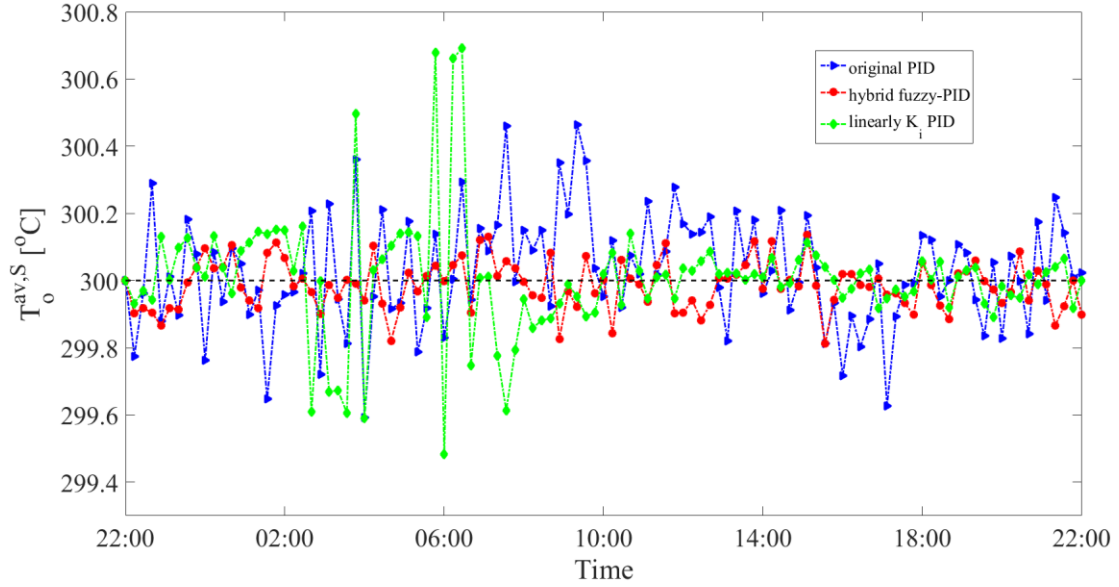


Fig. 19. Realistic daily simulation of the $T_0^{av,S}$ evolution with the optimized hybrid fuzzy-PID controller (line with dots), with the linear variable K_i PID (line with diamonds) and with the original PID (line with triangles), at the normal operating conditions

C. Sensors Failure Scenarios

Adaptivity of the CPS to sensors failures has also been analyzed for demonstration purpose. Indeed, transients due to wrong measurements can affect the NPP dynamics, since any transient, even though on the secondary side, can propagate to the primary coolant system and, eventually to the reactor core [36]. With no intention to treat a full spectrum of failures, for the purpose of exemplification, two failures are considered:

- 1) Drift: at 06:00, $T_a^{in,S}$ drifts up to 28°C that is reached at roughly 06:07, and, then, drops back to the normal environmental condition at 06:10 (see Fig. 20(b));
- 2) Bias: within the time interval between 18:00 and 18:05, the measured $T_a^{in,S}$ is biased (because of a sensor bias or a malicious cyber attack) with an increased noise distributed as $U(-10, 1)^\circ\text{C}$ (see Fig. 20(c)).

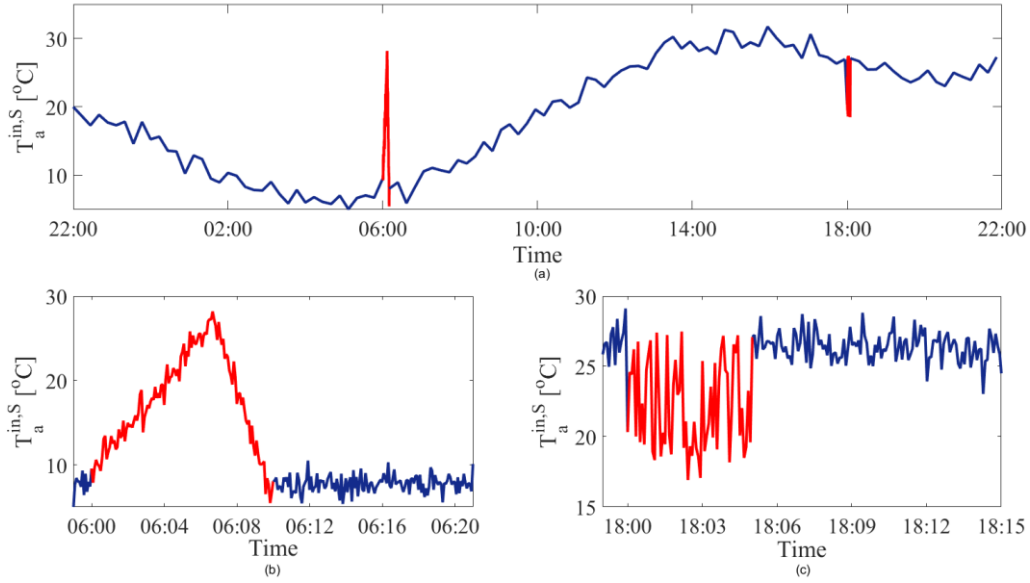


Fig. 20. Sketch of the assumed sensors failures: (a) the assumed daily $T_a^{in,S}$ profile; (b) drift of $T_a^{in,S}$ within [06:00, 06:10]; (c) bias of $T_a^{in,S}$ within [18:00, 18:05]

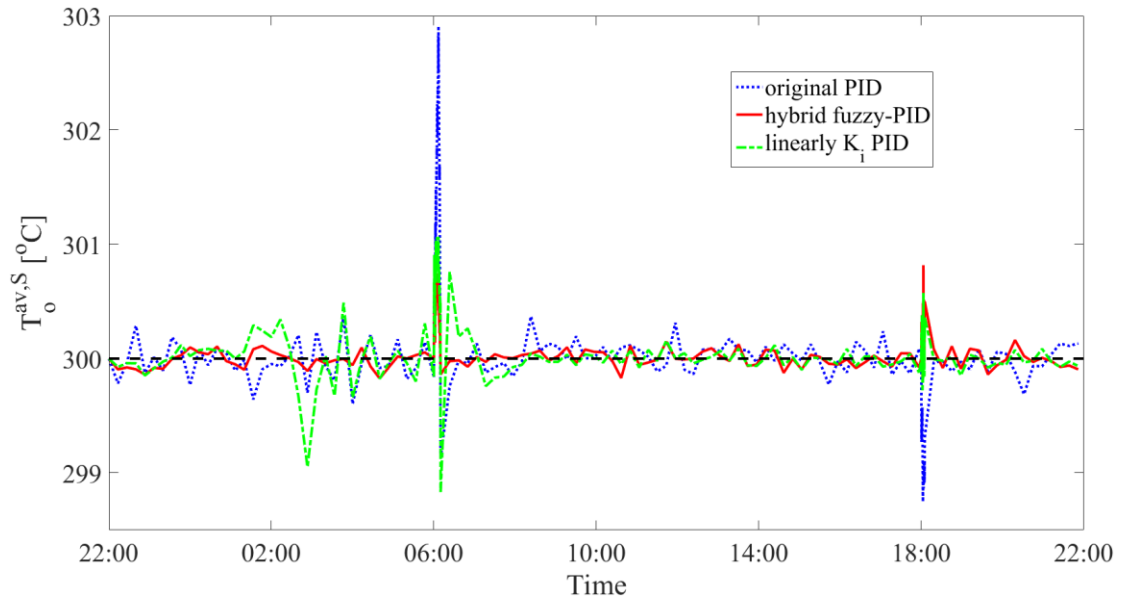


Fig. 21. $T_0^{av,S}$ evolution with the optimized hybrid fuzzy-PID controller (continuous line), with the linear variable K_i PID (dashed line) and with the original PID (dotted line), under the assumed sensors failure conditions

Fig. 21 shows that under the two assumed $T_a^{in,S}$ sensors failures, the $T_0^{av,S}$ fluctuations provided by the linear variable K_i PID (dashed line) and by the original PID (dotted line) are larger than those of the hybrid fuzzy-PID controller (continuous line).

Let us focus on the local details of the $T_o^{av,S}$ evolutions within the two spans of transients, including the post-transient adaptivity periods. For the transient periods and post-transient periods, the average drifts obtained with the hybrid fuzzy-PID controller are much smaller than those obtained with the original PID (see TABLE II).

TABLE II

Comparison of the average drifts of the $T_o^{av,S}$ evolutions with the hybrid fuzzy PID, the linear variable K_i PID and the original PID, under the transient operating conditions

Transient type	Time interval	\bar{e}_{SS} (°C)		
		Original PID	Variable K_i PID	Hybrid fuzzy-PID
Drift	[06:00, 06:10]	1.4040	0.7345	0.4268
	[06:10, 06:20]	1.2464	0.3363	0.1555
Bias	[18:00, 18:05]	0.8439	0.1852	0.2622
	[18:05, 18:15]	0.4327	0.0770	0.1533

Figs. 22 and 23 zoom the $T_o^{av,S}$ evolutions obtained from the simulations with the optimized hybrid fuzzy-PID controller, with the linear variable K_i PID and with the original PID, under the transients (a) and (b), respectively. The original PID cannot adaptively respond to the rapid fluctuations of the environmental conditions (dotted line with triangle) and requires roughly 600s (from 06:10 to 06:20) to recover the system stability under sensor drift circumstances (see Fig. 22). The linearly variable K_i PID can rapidly respond to the transient and recover the system stability from the drift failure scenario as shown in Fig. 21, however, $T_o^{av,S}$ largely deviates from the reference value with an \bar{e}_{SS} equal to 0.7345°C within the transient period (dashed line with diamonds) and an \bar{e}_{SS} equal to 0.3363°C within the recovery period. Despite an \bar{e}_{SS} equal to 0.4268°C within the transient period (continuous line with dots), $T_o^{av,S}$ can promptly recover the nominal temperature with an \bar{e}_{SS} equal to 0.1555°C, which is much smaller than that obtained from the original PID (1.2464°C) (see TABLE II).

Similarly, the original PID poorly adapts to the sensor bias failure scenario (see Fig. 23, dotted line with triangle) and yields an \bar{e}_{SS} equal to 0.8439°C. Despite larger \bar{e}_{SS} during both the transient and the post-transient periods (\bar{e}_{SS} equal to 0.2622°C and

0.1533°C, respectively (see TABLE II)) compared with the linearly variable K_i PID controller, the proposed hybrid fuzzy-PID can still rapidly respond to the transient and recover the system from the sensor bias failure.

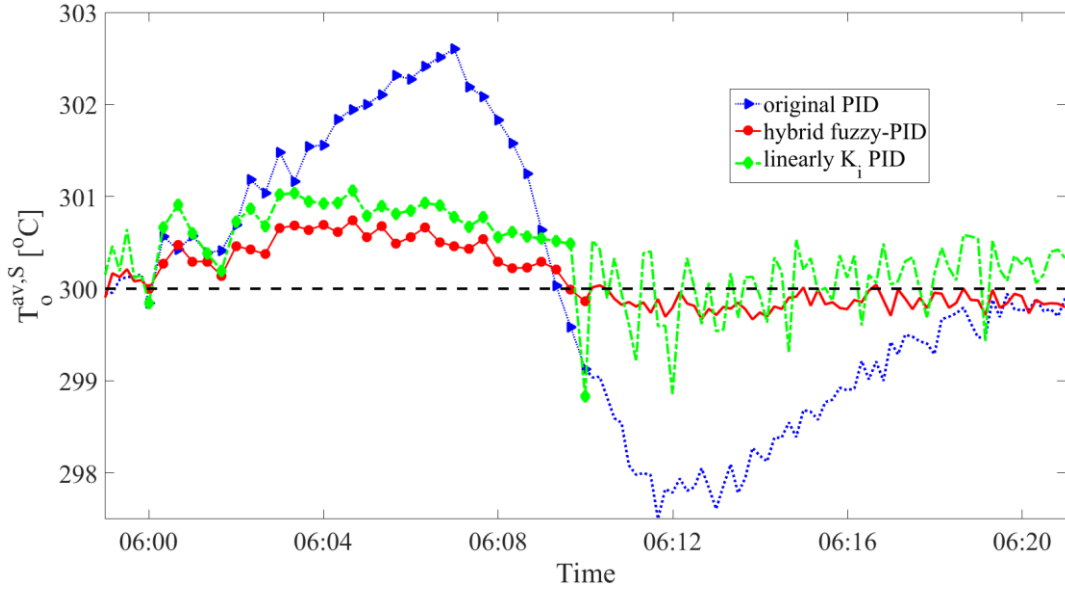


Fig. 22. Comparison of the $T_o^{av,S}$ evolution with the optimized hybrid fuzzy-PID controller (continuous line), with the linear variable K_i PID (dashed line) and with the original PID (dotted line), under the sensor drift failure scenario

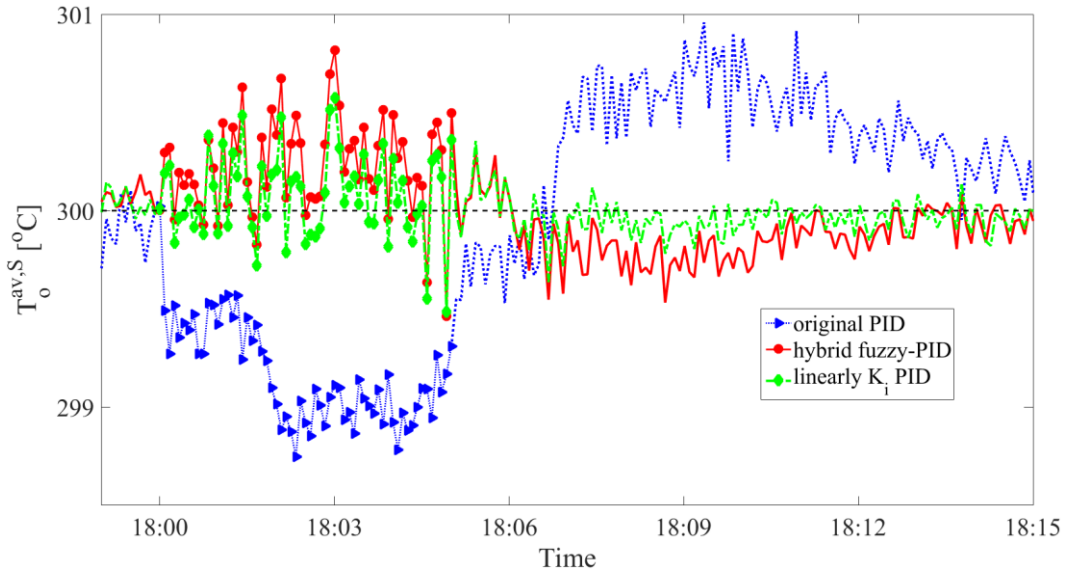


Fig. 23. Comparison of the $T_o^{av,S}$ evolution with the optimized hybrid fuzzy-PID controller (continuous line), with the linear variable K_i PID (dashed line) and with the original PID (dotted line), under the sensor bias failure scenario

In conclusion, the proposed hybrid fuzzy-PID controller achieves less average

drifts \bar{e}_{SS} than other controllers, in realistic scenarios of daily cycles of inlet air temperatures and also in accidental transients due to wrong sensor measurements, in the end preventing the system dynamics from deviating from steady-state beyond safety limits.

V. CONCLUSIONS

A hybrid fuzzy-PID control strategy is proposed for controlling a Cyber-Physical System (CPS), under varying environmental conditions. Environmental parameters values are fed to the hybrid fuzzy-PID, which adaptively optimizes the PID gains for controlling the physical variables of interest so as to keep them within preset safety bounds and, thus, improve the real-time performance of CPSs. To increase the transparency of the fuzzy reasoning underpinning the control, a grid-type fuzzy partitioning approach is proposed for optimizing the fuzzy sets and fuzzy rules, which are learnt from examples of input-output data pairs.

For illustration purposes, the proposed controller is applied for the control of the LBE-XADS secondary side average temperature, by adaptively responding to changes of the inlet air temperature. Results demonstrate the benefits that the hybrid fuzzy-PID controller offers over traditional PIDs, effectively reducing the resettling time of the system response to changes in the inlet air temperature.

APPENDIX. SENSITIVITY ANALYSIS FOR THE IDENTIFICATION OF THE OPTIMAL NUMBER OF FUZZY SETS

In this analysis, a number $S=3, 5, 7, 9, 11$ of input membership functions T are tested, resulting in S output fuzzy sets, according to the procedure of Section III.B.b. The 18 samples of Section IV.A are run with the LBE-XADS model equipped with the resulting hybrid fuzzy-PID controllers. Results are shown in Fig. A.1: Fig. A.1(a) shows that the hybrid fuzzy-PIDs with $S=7$ and $S=11$ perform better than the others, because A_{SP} is below 300 (at any environmental temperature), whereas the others cannot. Fig. A.1(b) shows that $S=7$ is to be preferred, because \bar{e}_{SS} is smaller than 0.07°C and

acceptable (when $T_{a,C}^{in,S}$ larger than 8°C).

The compromise between performance, robustness and computational demand has, thus, led us to set up a hybrid fuzzy-PIDs with 7 sets of membership functions.

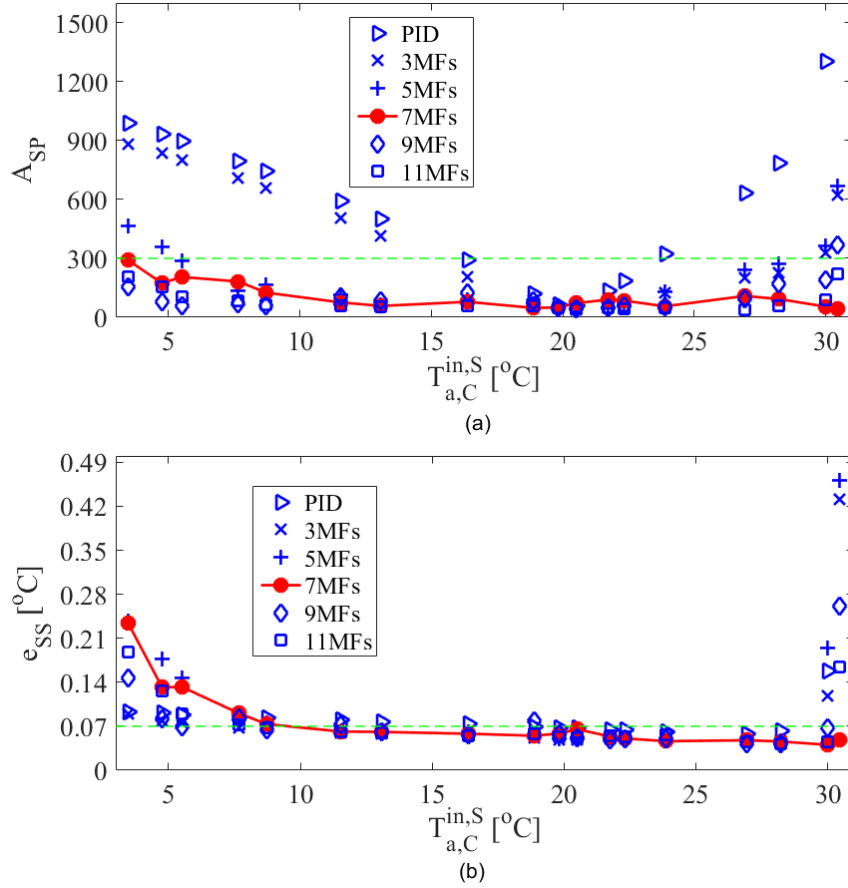


Fig. A.1. SA results of the samples obtained from the hybrid fuzzy-PID controller with different sets of membership functions and from the original PID: (a) settling area A_{SP} ; (b) steady-state drift \bar{e}_{SS}

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